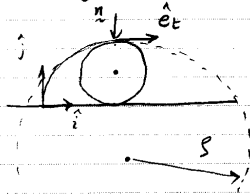


Solutions by Bqsaant Sharma April 29 '02

① (particle on the top of a rolling wheel)

A particle on the top of a rolling wheel (with speed  $v$  and radius  $R$ ) has position given by



$$\mathbf{r}(t) = \left( vt - R \sin \frac{vt}{R} \right) \hat{i} + R \left( 1 - \cos \frac{vt}{R} \right) \hat{j}$$

where at  $t=0$  we have the origin. When the particle is at its highest point  $vt = \pi R$ , so

$$\mathbf{v} = 2v \hat{i}, \quad \mathbf{a} = -\frac{v^2}{R} \hat{j}$$

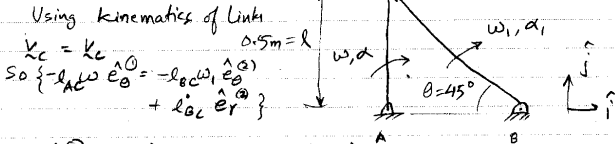
so  $\hat{e}_t = \hat{i}, \quad \hat{e}_n = -\hat{j}$

So  $\rho = \frac{|\mathbf{v}|^2}{|\mathbf{a}|} = \frac{(2v)^2}{(v^2/R)} = 4R$

So  $\rho_{oscillating} = 4R$

② 9.35

Given  $\omega = 3 \text{ rad/s}, d = 2 \text{ rad/s}^2$



Using kinematics of links

$$\mathbf{v}_C = \mathbf{v}_A + \boldsymbol{\omega} \times \mathbf{r}_{AC}$$

$$\mathbf{v}_C = -l\omega \hat{e}_\theta + l\dot{\theta} \hat{e}_t$$

$\mathbf{v}_C \cdot \hat{e}_r = 0 \Rightarrow -l\omega \cos \theta + l\dot{\theta} = 0$   
 $\Rightarrow \dot{\theta} = \omega \cos \theta$

$\mathbf{a}_C = \mathbf{a}_A + \dot{\boldsymbol{\omega}} \times \mathbf{r}_{AC} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}_{AC}) + 2\dot{\boldsymbol{\omega}} \times \mathbf{r}_{AC} + \boldsymbol{\omega} \times \dot{\mathbf{r}}_{AC}$   
 $\Rightarrow -l\omega^2 \hat{e}_r + l\dot{\omega} \hat{e}_\theta + 2l\dot{\omega} \hat{e}_t + l\omega \dot{\theta} \hat{e}_t - l\omega^2 \hat{e}_r = (l\dot{\omega} - l\omega^2) \hat{e}_\theta + (2l\dot{\omega} + l\omega \dot{\theta}) \hat{e}_t$

$\mathbf{a}_C \cdot \hat{e}_r = 0 \Rightarrow l\dot{\omega} - l\omega^2 = 0$   
 $\Rightarrow \dot{\omega} = \omega^2$   
 $\Rightarrow \omega = \frac{1}{2} \text{ rad/s}$

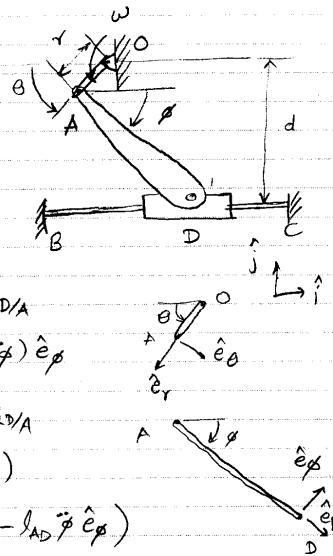
Finally  $\mathbf{a}_C \cdot \hat{e}_\theta = -2l\omega \dot{\omega} + l\omega^2 \left( \frac{1}{\sqrt{2}} \right) + l\omega \left( \frac{1}{\sqrt{2}} \right)$   
 $= \left[ -2 \left( \frac{3}{2\sqrt{2}} \right) \left( \frac{3}{2} \right) + \frac{1}{2} (3)^2 \left( \frac{1}{\sqrt{2}} \right) + \frac{1}{2} (2) \left( \frac{1}{\sqrt{2}} \right) \right] \frac{m}{s^2}$

So  $\alpha_1 = \frac{\sqrt{2}}{2} (9 - 9 + 2) \text{ rad/s}^2 = 1 \text{ rad/s}^2$

③ 9.41

Given  $r = 50 \text{ mm}$   
 $\dot{\theta} = \omega = 30 \text{ rad/s}$   
 $l_{AD} = 250 \text{ mm}$   
 $d = 150 \text{ mm}$

Using kinematics of mechanisms,



$\mathbf{v}_D = \mathbf{v}_O + \mathbf{v}_{NO} + \mathbf{v}_{D/A}$   
 $= r\dot{\theta} \hat{e}_\theta + l_{AD} \dot{\phi} \hat{e}_\phi$

$\mathbf{a}_D = \mathbf{a}_O + \mathbf{a}_{AO} + \mathbf{a}_{D/A}$   
 $= (-r\dot{\theta}^2 \hat{e}_r + r\ddot{\theta} \hat{e}_\theta) + (-l_{AD} \dot{\phi}^2 \hat{e}_r - l_{AD} \ddot{\phi} \hat{e}_\phi)$

So  $\mathbf{v}_D = r\dot{\theta} \hat{e}_\theta - l_{AD} \dot{\phi} \hat{e}_\phi$  (1)

$\mathbf{a}_D = -r\dot{\theta}^2 \hat{e}_r - l_{AD} (\dot{\phi}^2 \hat{e}_r + \ddot{\phi} \hat{e}_\phi)$  (2)

at the instant of interest  $\theta = \frac{\pi}{2}$

So  $\phi = \sin^{-1} \left( \frac{d-r}{l_{AD}} \right)$  (3)

Due to the constraint of slider at D

$\mathbf{v}_D \cdot \hat{j} = 0, \quad \mathbf{a}_D \cdot \hat{j} = 0$  (4)

(1)  $\Rightarrow r\dot{\theta} (\hat{e}_\theta \cdot \hat{j}) - l_{AD} \dot{\phi} (\hat{e}_\phi \cdot \hat{j}) = 0$

or  $\dot{\phi} = 0$  (since  $\cos \phi \neq 0, l_{AD} \neq 0$ )

Similarly

(2)  $\Rightarrow -r\dot{\theta}^2 \hat{e}_r \cdot \hat{j} - l_{AD} (\dot{\phi}^2 \hat{e}_r \cdot \hat{j} + \ddot{\phi} \hat{e}_\phi \cdot \hat{j}) = 0$

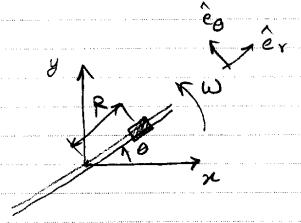
or  $-r\dot{\theta}^2 (-1) - l_{AD} (\dot{\phi} \cos \phi) = 0$

$\Rightarrow \ddot{\phi} = \frac{r\dot{\theta}^2}{l_{AD} \cos \phi}$

using (3)  $\ddot{\phi} = \frac{r\dot{\theta}^2}{\sqrt{l_{AD}^2 - (d-r)^2}} \approx 96.4 \text{ rad/s}^2$

④ 10.8

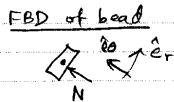
Assume  $\omega = \omega_0$  is constant, the bead (mass) is free to slide; at  $t=0$  the bead is at 1 ft from the origin and  $\frac{dR}{dt} = 0$ .



a)  $\mathbf{v}_{bead} = \dot{R} \hat{e}_r + R \dot{\theta} \hat{e}_\theta$

$\mathbf{a}_{bead} = (\ddot{R} - R\dot{\theta}^2) \hat{e}_r + (2\dot{R}\dot{\theta} + R\ddot{\theta}) \hat{e}_\theta$

by LMB:  $\{ m \mathbf{a}_{bead} = N \hat{e}_\theta \}$



So  $\{ \} \cdot \hat{e}_r \Rightarrow m a_{bead} \cdot \hat{e}_r = 0$

$\Leftrightarrow \ddot{R} - R\dot{\theta}^2 = 0$

$\Leftrightarrow \ddot{R} = R\omega_0^2$  (1)

b) Since  $\dot{\theta} = \omega_0 = \text{const.}$

$\theta = \omega_0 t$  (at  $t=0, \theta=0$ )

So  $\frac{d^2 R}{dt^2} = \frac{d^2 R}{d\theta^2} \left( \frac{d\theta}{dt} \right)^2 = \omega_0^2 \frac{d^2 R}{d\theta^2}$

So (1)  $\Leftrightarrow \frac{d^2 R(\theta)}{d\theta^2} = R(\theta)$  (2)

Solving (2)  $R(\theta) = \frac{1}{2} (e^\theta + e^{-\theta})$  ft  
 (use  $R(0) = 1 \text{ ft}, \dot{R}(0) = 0$ )

c) after 1 revolution  
 $R_1 = R(2\pi) = \frac{1}{2} (e^{2\pi} + e^{-2\pi}) \text{ ft} \approx 267.7 \text{ ft}$

after 2 revolutions  
 $R_2 = R(4\pi) = \frac{1}{2} (e^{4\pi} + e^{-4\pi}) \text{ ft} \approx 1.93 \times 10^5 \text{ ft}$

d)  $\dot{R}(t) = \frac{dR(\theta)}{d\theta} \cdot \frac{d\theta}{dt} = \omega_0 \frac{dR(\theta)}{d\theta} = \frac{\omega_0}{2} (e^\theta - e^{-\theta})$ , using  $\omega_0 = 2\pi \text{ rad/s}$   
 $\therefore \dot{R}_1 = \dot{R}(1) = \frac{2\pi}{2} (e^{2\pi} - e^{-2\pi}) \frac{\text{ft}}{\text{s}} \approx 1682.3 \text{ ft/s}$

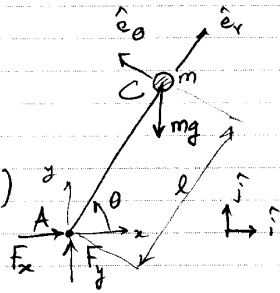
e)  $K E_{bead} = \frac{1}{2} m \dot{R}_1^2$   
 $\approx (283 \times 10^6) \times \text{mass} \left( \frac{\text{ft}}{\text{s}} \right)^2$  [use  $v_1 = \dot{R}(1)$ ]

⑤ 10.24 (Balancing a broom)

Kinematics:

$$\begin{aligned} \mathbf{r}_C &= \mathbf{r}_A + \mathbf{r}_{C/A} \\ &= a\hat{i} + (-l\dot{\theta}^2\hat{e}_r + l\ddot{\theta}\hat{e}_\theta) \end{aligned}$$

by  $\underline{AMB/A}$ :  $\underline{\dot{H}}_A = \underline{\Sigma M}_A$



$$\Leftrightarrow l\hat{e}_r \times m\mathbf{a}_C = -mgl\cos\theta\hat{k}$$

$$\Leftrightarrow \{lm(a\hat{e}_r \times \hat{i} + l\ddot{\theta}\hat{e}_r \times \hat{e}_\theta) = -mgl\cos\theta\hat{k}\}$$

$$\{3.\hat{k} \Rightarrow ml(-a\sin\theta + l\ddot{\theta}) = -mgl\cos\theta$$

$$\Rightarrow \ddot{\theta} = \frac{1}{l}(-g\cos\theta + a\sin\theta)$$

by  $\underline{LMB}$ :  $\{m\mathbf{a}_C = F_x\hat{i} + F_y\hat{j} - mg\hat{j}\}$

$$\begin{aligned} \{3.\hat{i} \Rightarrow F_x &= m\mathbf{a}_C \cdot \hat{i} \\ &= m(a - l\dot{\theta}^2\hat{e}_r \cdot \hat{i} + l\ddot{\theta}\hat{e}_\theta \cdot \hat{i}) \\ &= m(a - l\dot{\theta}^2\cos\theta + l\ddot{\theta}(-\sin\theta)) \\ &= m(a - l\dot{\theta}^2\cos\theta + g\sin\theta\cos\theta - a\sin^2\theta) \end{aligned}$$

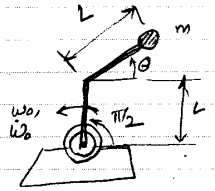
$$\begin{aligned} \{3.\hat{j} \Rightarrow F_y &= mg + m(-l\dot{\theta}^2\hat{e}_r \cdot \hat{j} + l\ddot{\theta}\hat{e}_\theta \cdot \hat{j}) \\ &= m(g - l\dot{\theta}^2\sin\theta + (-g\cos\theta + a\sin\theta)\cos\theta) \\ &= m(g - l\dot{\theta}^2\sin\theta - g\cos^2\theta + a\sin\theta\cos\theta) \end{aligned}$$

So the force of the hand is

$$\begin{aligned} \underline{\mathbf{F}} &= m(a - l\dot{\theta}^2\cos\theta + g\sin\theta\cos\theta - a\sin^2\theta)\hat{i} \\ &\quad + m(g - l\dot{\theta}^2\sin\theta - g\cos^2\theta + a\sin\theta\cos\theta)\hat{j} \end{aligned}$$

⑥ 10.26

A motor at O turns at  $\omega_0, \dot{\omega}_0$ . At the end of a stick connected to this motor is a frictionless hinge attached to another massless stick. Both sticks have length L. At the end of the second stick is a mass m. What is  $\ddot{\theta}$ ?

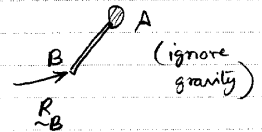


$\underline{AMB/B}$ :  $\underline{\Sigma M}_B = \underline{\dot{H}}_B$

FBD of 2<sup>nd</sup> stick with mass

$$\underline{\Sigma M}_B = 0$$

$$\underline{\dot{H}}_B = \underline{\mathbf{r}}_{A/B} \times m\mathbf{a}_A$$



Now using kinematics

$$\begin{aligned} \mathbf{r}_A &= \mathbf{r}_B + \mathbf{r}_{A/B} \\ &= \omega_0 \times (\omega_0 \times \mathbf{r}_{B/O}) + \dot{\omega}_0 \times \mathbf{r}_{B/O} \\ &\quad + \dot{\theta} \hat{k} \times (\dot{\theta} \hat{k} \times \mathbf{r}_{A/B}) + \ddot{\theta} \hat{k} \times \mathbf{r}_{A/B} \\ &= (-\omega_0^2 L - \dot{\theta}^2 L \cos\theta - \ddot{\theta} L \sin\theta)\hat{i} \\ &\quad + (-\omega_0^2 L - \dot{\theta}^2 L \sin\theta + \ddot{\theta} L \cos\theta)\hat{j} \end{aligned}$$

From  $\underline{AMB/B}$  above,  $\underline{\dot{H}}_B = 0$

$$\Rightarrow \underline{\mathbf{r}}_{A/B} \times m\mathbf{a}_A = 0$$

$$\Rightarrow mL(\cos\theta\hat{i} + \sin\theta\hat{j}) \times [(-\omega_0^2 L - \dot{\theta}^2 L \cos\theta - \ddot{\theta} L \sin\theta)\hat{i} + (-\omega_0^2 L - \dot{\theta}^2 L \sin\theta + \ddot{\theta} L \cos\theta)\hat{j}] = 0$$

$$\Rightarrow \{mL^2(-\omega_0^2 \cos\theta - \dot{\theta}^2 \sin\theta \cos\theta + \ddot{\theta} \cos^2\theta + \omega_0^2 \sin\theta + \dot{\theta}^2 \sin\theta \cos\theta + \ddot{\theta} \sin^2\theta)\hat{k} = 0\}$$

$$\{3.\hat{k} \Rightarrow \ddot{\theta} = \omega_0^2 \cos\theta - \dot{\omega}_0 \sin\theta\}$$