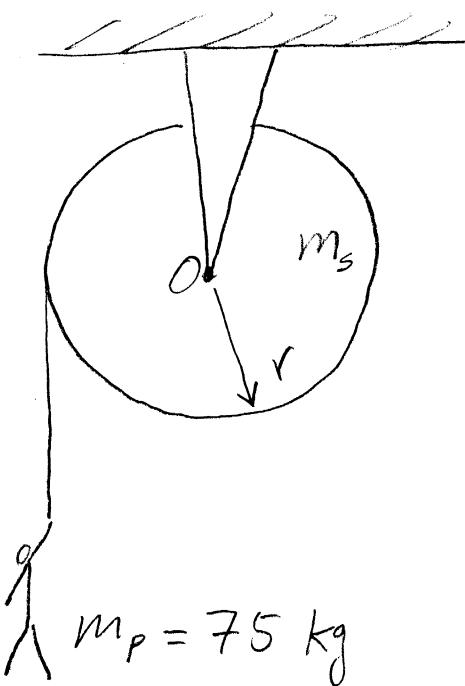


ENGRD/TAM 203

HW24 (Assigned on April 20, due on April 27)
Solution by Dennis Yang

7.2.33

The spool has a mass of 80 kg and a radius of 1.1 m. The axle/spool interface provides a friction so that 11 N.m of torque is needed to start the spool turning and remains at this level once the spool is rotating. The person's mass is 75 kg. What is his vertical acceleration ? (neglect the mass of the cable)



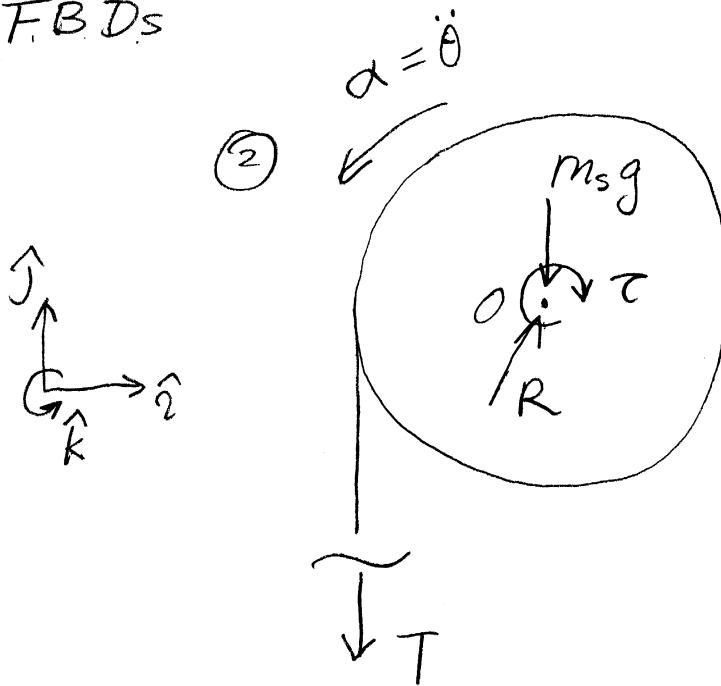
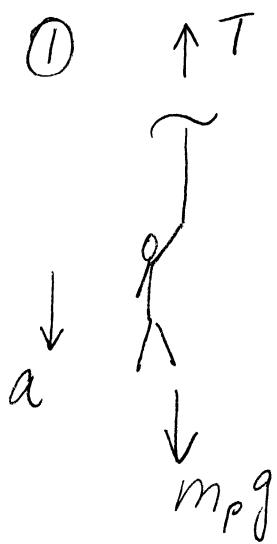
$$m_s = 80 \text{ kg}$$

$$r = 1.1 \text{ m}$$

$$m_p = 75 \text{ kg}$$

Solution

FBDs



Since we neglect the mass of the cable,
the tension is the same along the vertical part
of the cable .
(c)

Because the cable is inextensible , $a = \alpha r = \ddot{\theta}r$

For ① :

$$\sum \vec{F}_i = m \vec{a} \Rightarrow m_p g (-\hat{j}) + T(\hat{j}) = m_p a(-\hat{j}) \quad (a)$$

For ② :

$$\sum \vec{M}_{i_0} = I_{i_0} \vec{\alpha} \Rightarrow Tr \hat{k} + \tau(-\hat{k}) = I_{i_0} \ddot{\theta} \hat{k} \quad (b)$$

3.

$$(a) \cdot \hat{j} \Rightarrow -m_p g + T = -m_p a \quad (e)$$

$$(b) \cdot \hat{r} \Rightarrow Tr - \tau = I_{10} \ddot{\theta} = \underbrace{\frac{m_s r^2}{2}}_{I_{10} \text{ for circular plate}} \ddot{\theta} \quad (d)$$

I₁₀ for circular plate

$$(d) \Rightarrow Tr - \tau = \frac{m_s r}{2} a$$

$$\Rightarrow T - \frac{\tau}{r} = \frac{m_s}{2} a \quad (f)$$

$$(f) - (e) \Rightarrow m_p g - \frac{\tau}{r} = \left(\frac{m_s}{2} + m_p\right)a$$

$$\Rightarrow a = \frac{m_p g - \tau_r}{m_s/2 + m_p}$$

$$a = \frac{75 \text{ kg} \cdot 9.8 \text{ m/s}^2 - 11 \text{ N} \cdot \text{m} / 1.1 \text{ m}}{80 \text{ kg}/2 + 75 \text{ kg}}$$

$$a \approx 6.30 \text{ m/s}^2$$

$$\vec{a} \approx 6.30 \text{ m/s}^2 (-\hat{j})$$

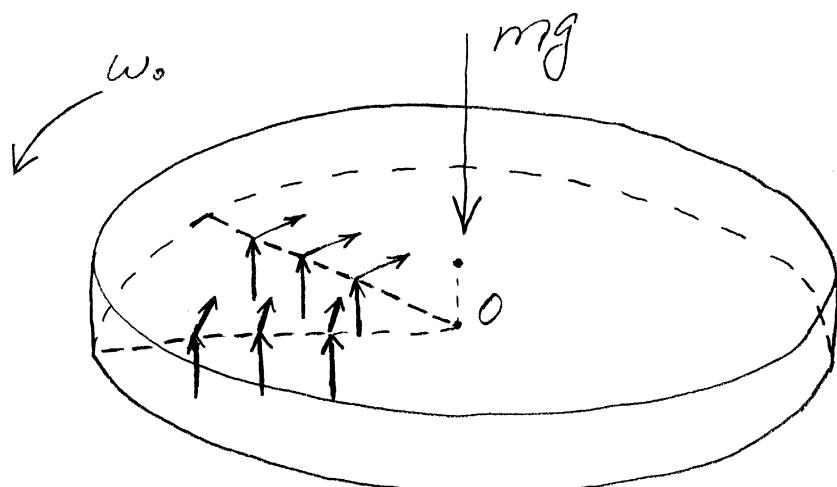


7.2.45

A uniform disc of radius R is spun at an angular speed ω_0 and then carefully placed, flat side down, on a horizontal surface. How long will the disc be rotating on the surface if the coefficient of dynamic friction is μ ?
(Assume the pressure exerted by the disc on the surface is uniform.)

Solution

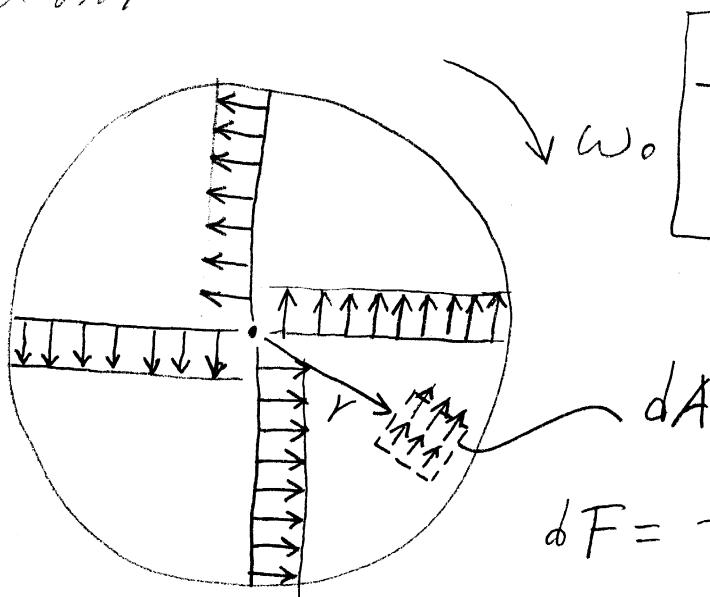
FBD



Note: the normal force and the friction are both uniformly distributed at the bottom of the disc, and the above drawing only gives a representative illustration.

Now, just look at tangential tractions (stresses) on the bottom

Assume the total mass of the disc is "m"



$$dF = \tau dA$$

$$= \mu dA$$

$$= \frac{mg}{A} \mu dA$$

$$\vec{M} = \iint_{\text{bottom surface}} \vec{r} \times \vec{F} \\ dF = r \tau dA, (-\hat{k})$$

$$= \int_0^{2\pi} \int_0^R r \cdot \frac{mg}{A} M \cdot \underbrace{\int dA}_{\tau} \cdot r dr d\theta \cdot (-\hat{k})$$

$$= \frac{mg}{A} M \left[\int_0^{2\pi} d\theta \right] \cdot \left[\int_0^R r^2 dr \right] (-\hat{k})$$

$$= \frac{mg}{\pi R^2} M \cdot 2\pi \cdot \frac{R^3}{3} (-\hat{k}) \quad (A = \pi R^2)$$

$$\boxed{\vec{M} = \frac{2}{3} mg \mu R (-\hat{k})}$$

6.

$$\vec{M} = I \vec{\alpha} \Rightarrow \frac{2}{3} mg \mu R (-\hat{k}) = \boxed{\frac{MR^2}{2} \alpha \hat{k}} \quad (*)$$

I

$$(*) \cdot \hat{k} \Rightarrow -\frac{2}{3} mg \mu R = \frac{MR^2}{2} \alpha$$

$$\Rightarrow \alpha = -\frac{4}{3} \frac{g \mu}{R}$$

$$\omega = \omega_0 + \alpha t^*$$

$$\Rightarrow t^* = -\frac{\omega_0}{\alpha}$$

$$= -\frac{\omega_0}{-\frac{4}{3} \frac{g \mu}{R}}$$

$$t^* = \frac{3 \omega_0 R}{4 g M}$$

